



# Experimental investigations of the scaling of turbulence in pipes and channels

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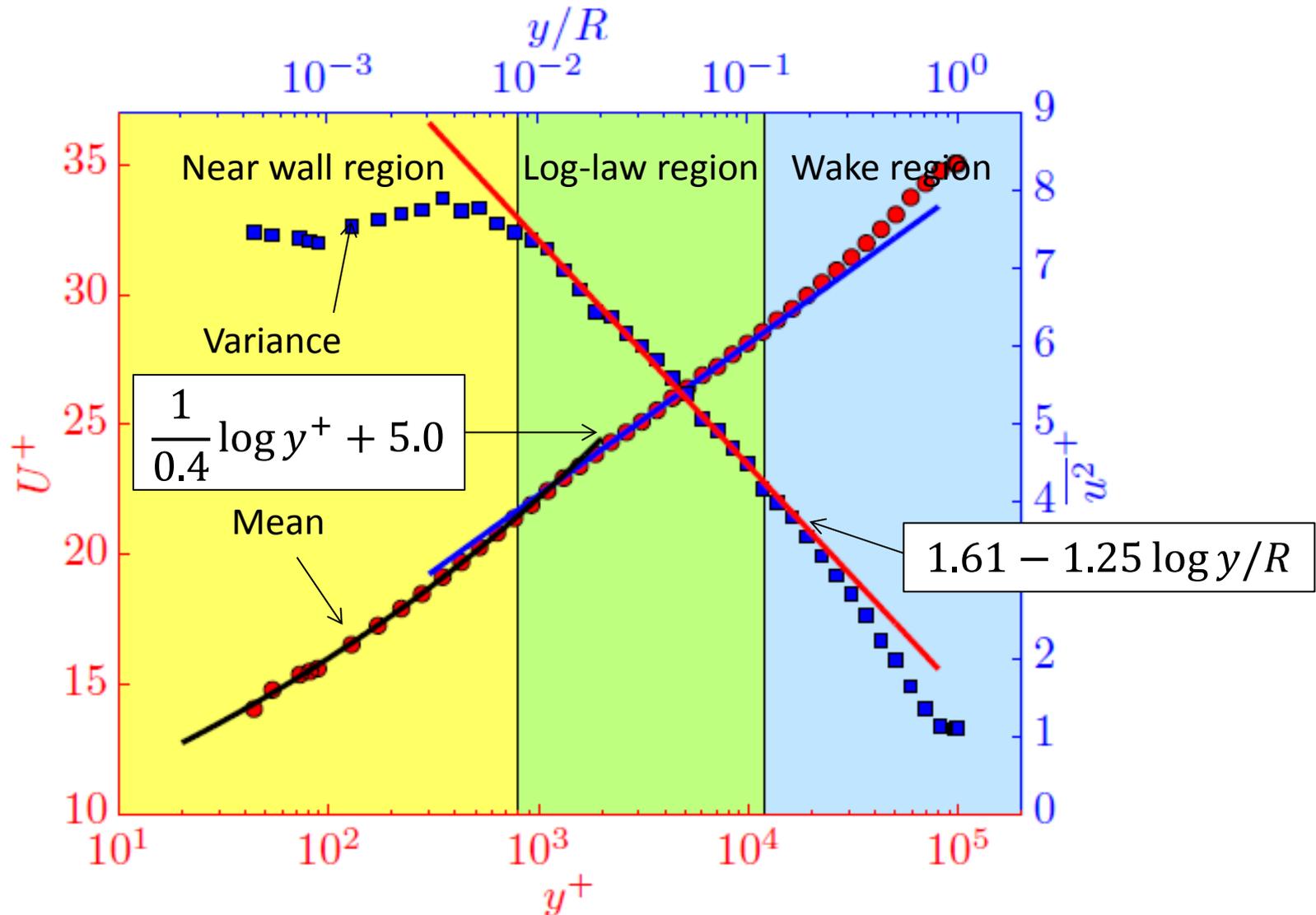
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***High Reynolds Number Boundary Layer Turbulence: Integrating Descriptions of Statistical Structure, Scaling and Dynamical Evolution***

University of New Hampshire

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# Duality between fluctuations and mean velocities



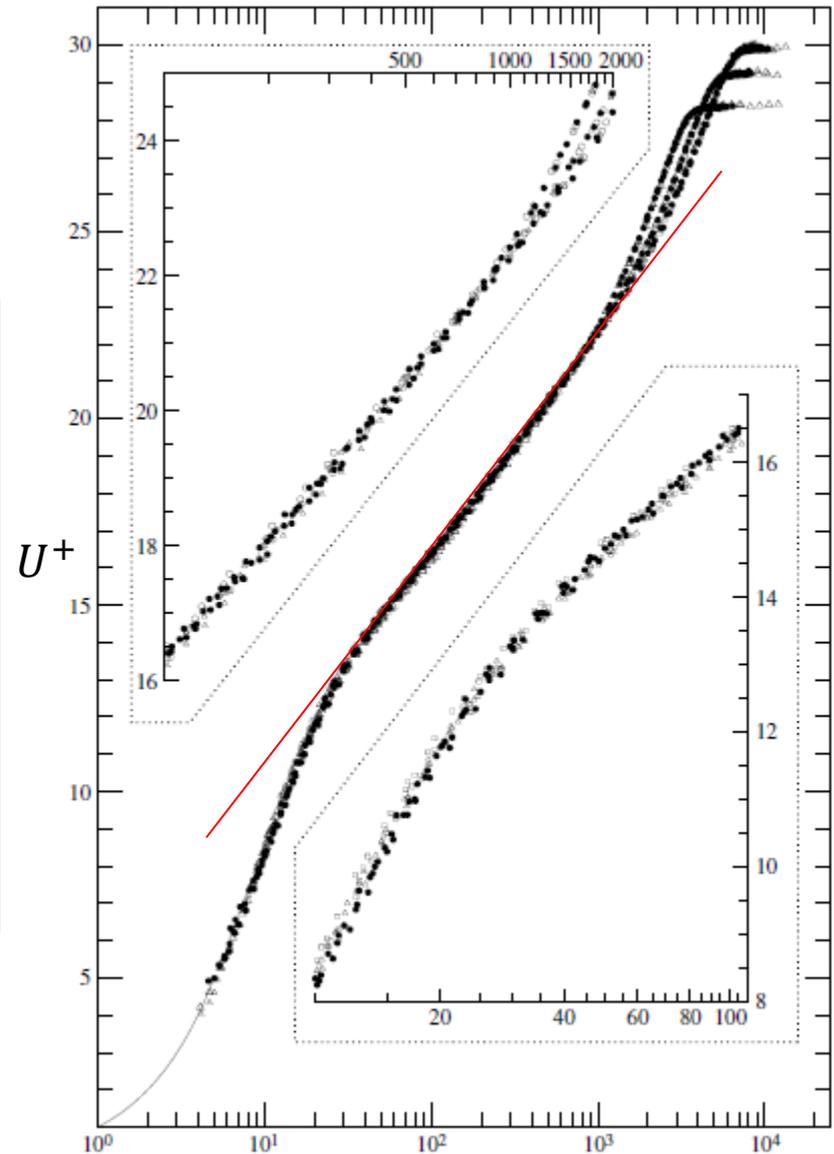
# Mean flow overlap argument

Inner:  $U^+ = f(y^+)$

Outer:  $\frac{U_\infty - U}{u_\tau} = g\left(\frac{y}{R}\right)$

$\longrightarrow U^+ = \frac{1}{\kappa} \ln(y^+) + B$

(log-law in the overlap region)



## Two problems with the classical scaling

- Only strictly valid at infinite Reynolds numbers.
- For mean velocities only, cannot explain the duality observed between the mean velocity and the fluctuations.

# Two problems with the classical scaling

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- Wosnik *et al.* (2000) extended it to finite Reynolds numbers by allowing the functions to vary with Reynolds number.

$$U^+ = f_i(y^+, R^+)$$

$$\frac{U_\infty - U}{u_\tau} = f_o(\bar{y}, R^+)$$

by introducing an intermediate variable  $\tilde{y} = y^+ R^{+ - n}$  and by differentiating with respect to  $R^+$  while keeping  $\tilde{y}$  constant (following George & Castillo 1997), they found that:

$$\bar{y} \frac{\partial f_o}{\partial \bar{y}} \Big|_{R^+} = \frac{1}{\kappa(R^+)} + \left[ \frac{\partial f_i(y^+, R^+)}{\partial \log(R^+)} \Big|_{y^+} - \frac{\partial f_o(\bar{y}, R^+)}{\partial \log(R^+)} \Big|_{\bar{y}} \right] = \frac{1}{\kappa(R^+)} + S_m$$

- If there exists a region in space where  $S_m = 0$ , the mean velocities will exhibit a the logarithmic law. Also note that  $S_m = \text{constant}$  would result in a logarithm.

- $\frac{1}{\kappa} = \frac{dU_{cl}^+}{d \log R^+}$

- $U^+ = \frac{1}{\kappa} \ln(y^+ + a^+) + B$

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- If  $S_m = 0$  then we recover the logarithmic law.

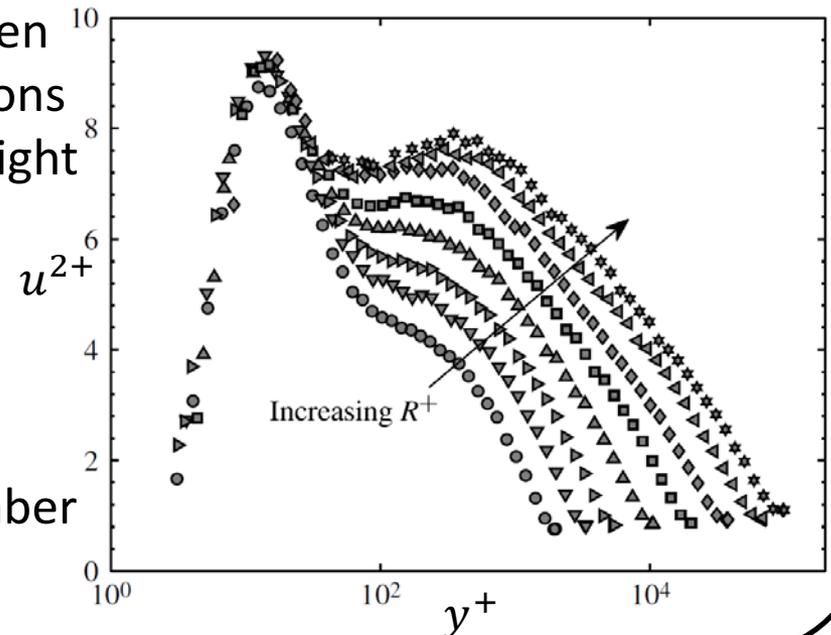
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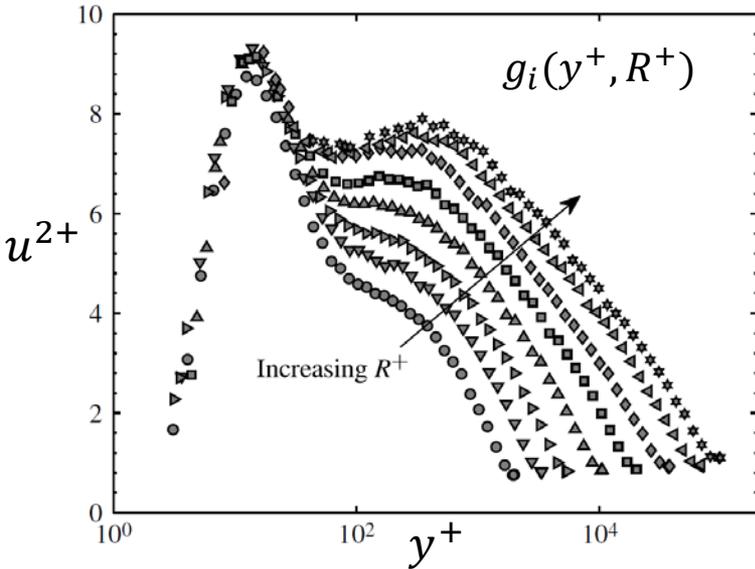
## Two problems with the classical scaling

- Only strictly valid at infinite Reynolds numbers.
- Does only work for the mean velocity, cannot explain the duality observed between the mean velocity and the fluctuations.

- The as good as perfect match between the logarithmic layer in the fluctuations and the mean suggests that there might be a matching theory.
- **Problem:** No obvious offset in the variances (centerline velocity for the mean velocities,  $(U_\infty - U)/u_\tau$ )
- There will always be a Reynolds number trend in the fluctuations.



# Reynolds number dependence in the fluctuations



Can extend the approach by Wosnik et al. (2000)

$$u^{2+} = g_i(y^+, R^+)$$

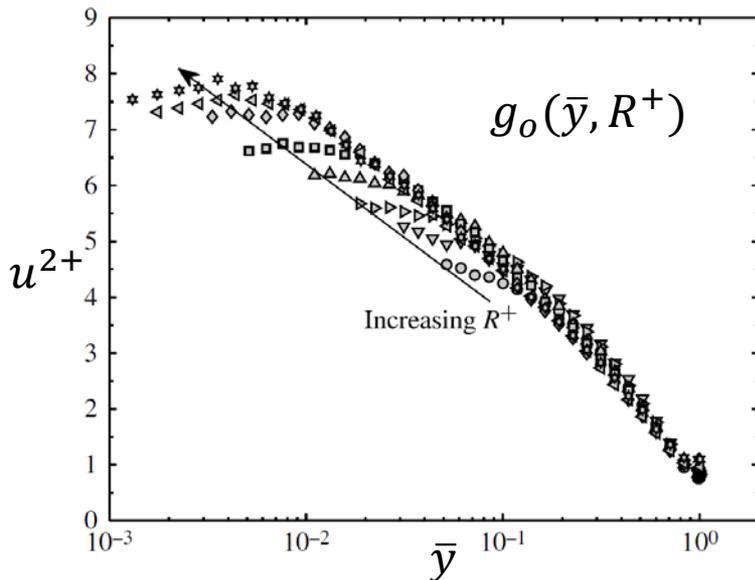
$$u^{2+} = g_o(\bar{y}, R^+)$$

introducing an intermediate variable  $\tilde{y} = y^+ R^{+n}$  and by differentiating with respect to  $R^+$  while keeping  $\tilde{y}$  constant, we find.

$$\bar{y} \frac{\partial g_o}{\partial \bar{y}} \Big|_{R^+} = \left[ \frac{\partial g_i(y^+, R^+)}{\partial \log(R^+)} \Big|_{y^+} - \frac{\partial g_o(\bar{y}, R^+)}{\partial \log(R^+)} \Big|_{\bar{y}} \right] = -S_f$$

If  $S_f$  is constant anywhere in space we can expect the profile to be logarithmic in the same region. And the slope of the logarithm will be  $-S_f$ .

$$u^{2+} = B_0 - S_f \log(\bar{y} + b^+)$$



# Sensitivity functions for the mean and the fluctuations

Evaluate the sensitivity functions by interpolating the data at  $Re_\tau = 98,000$  and  $Re_\tau = 37,000$  to match  $y^+$  and  $\bar{y}$  of  $Re_\tau = 68,000$  and evaluate the gradients in Reynolds number.

$$S_f = - \left[ \frac{\partial g_i(y^+, R^+)}{\partial \log(R^+)} \Big|_{y^+} - \frac{\partial g_o(\bar{y}, R^+)}{\partial \log(R^+)} \Big|_{\bar{y}} \right]$$

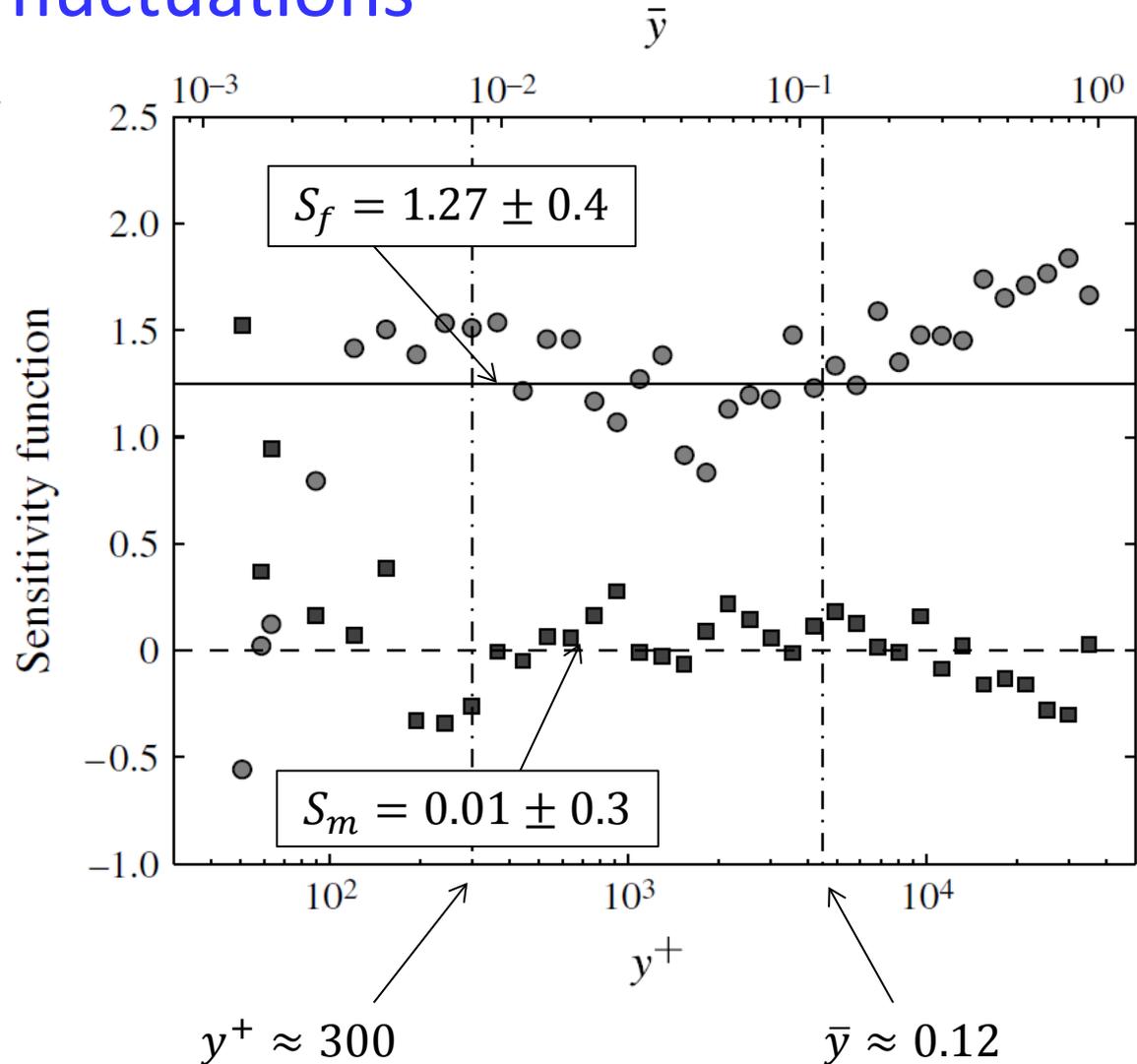
$$S_m = \left[ \frac{\partial f_i(y^+, R^+)}{\partial \log(R^+)} \Big|_{y^+} - \frac{\partial f_o(\bar{y}, R^+)}{\partial \log(R^+)} \Big|_{\bar{y}} \right]$$

Can expect:

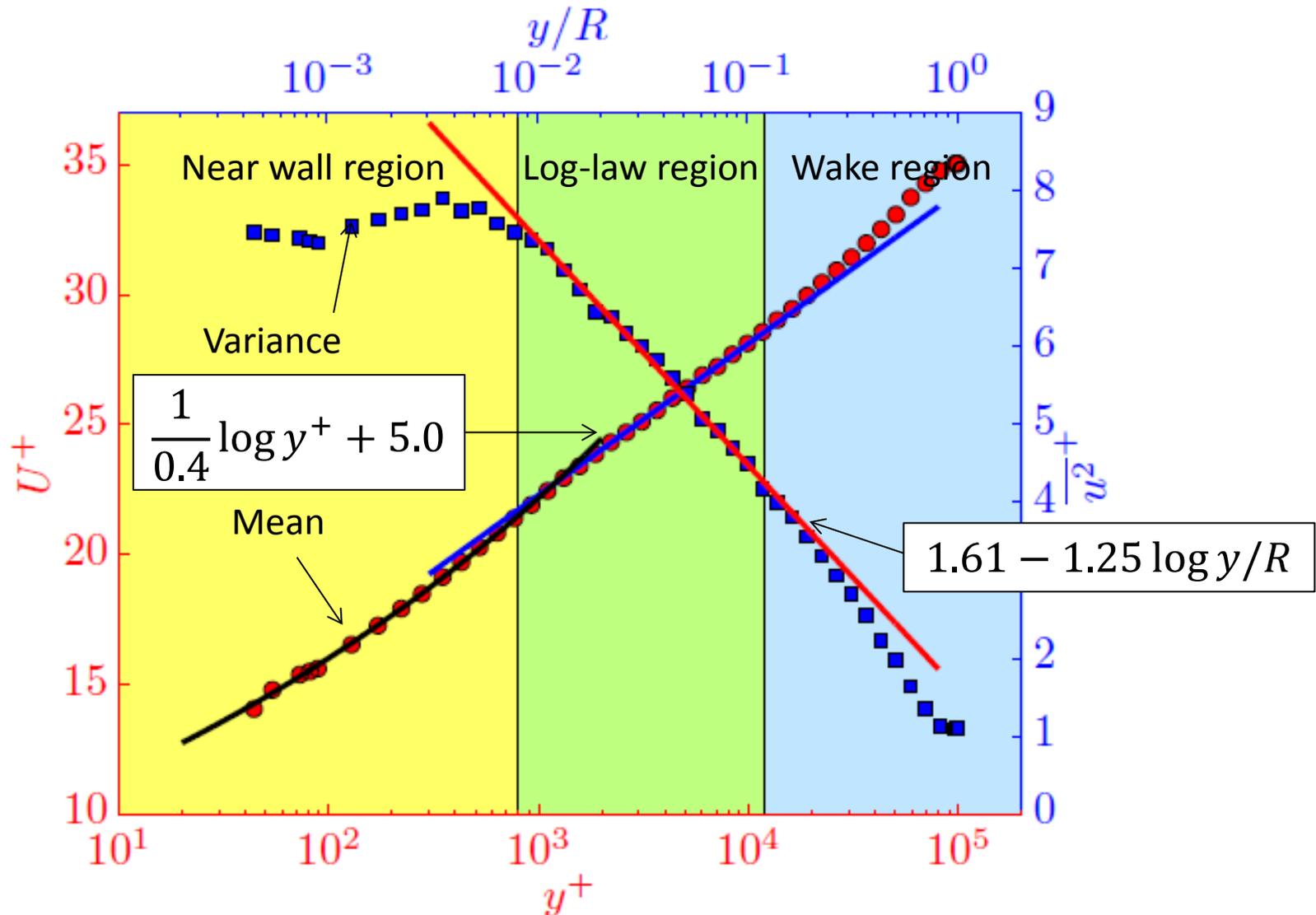
$$u^{2+} = B_1 - 1.27 \log(\bar{y} + \bar{b})$$

and

$$U^+ = \frac{1}{\kappa} \log(y^+ + a^+) + B$$



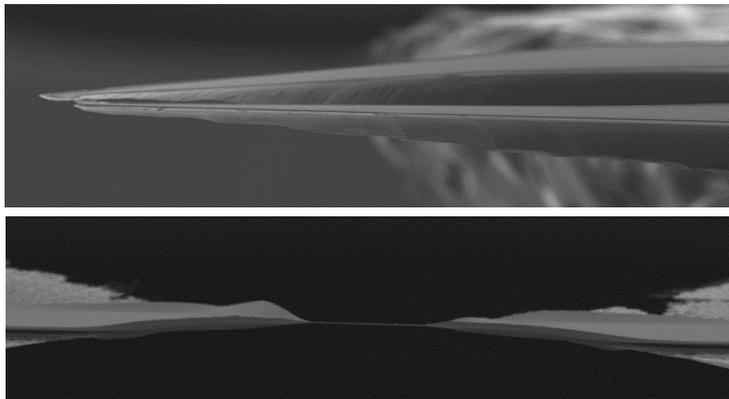
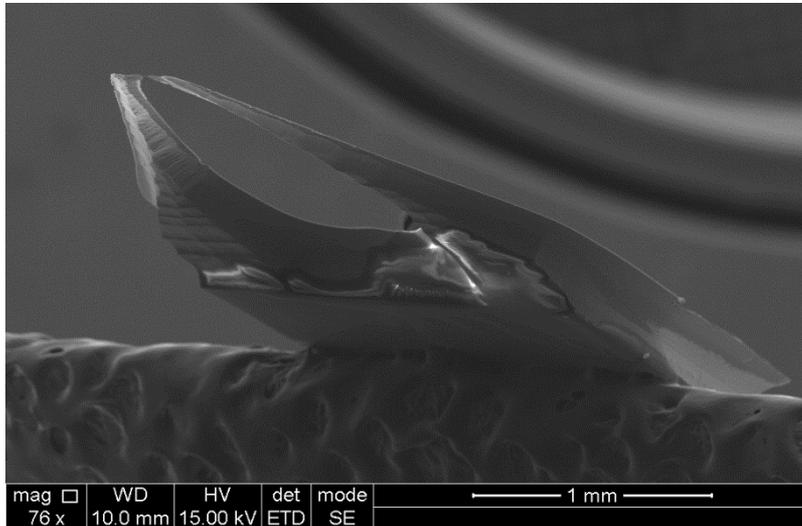
# Duality between fluctuations and mean velocities



# New sensors made it possible (together with a very special facility)

## Nano-Scale Thermal Anemometer Probe (NSTAP)

Bailey *et al.* (2010), Vallikivi *et al.* (2011), Vallikivi and Smits (under review)



## Superpipe



Zagarola and Smits (1997)

- The NSTAP is more than one order of magnitude smaller than regular hot wires (improved spatial resolution)
- Improved temporal resolution  $\sim 150\text{kHz}$
- **Well resolved turbulence measurements up to  $Re_\tau = 100,000$ .**

Similar scaling for passive scalars?  
What is needed for a detailed investigation?

# Scaling of passive scalar in wall bounded turbulence

$$\frac{\partial \theta}{\partial t} + u_j \frac{\partial \theta}{\partial x_j} = \alpha \nabla^2 \theta$$

$$\theta = \Theta + \theta'$$

$$u_j = U_j + u'_j$$

$U, \Theta$ : mean components

$u'_j, \theta'$ : fluctuations

$\alpha$ : scalar diffusivity

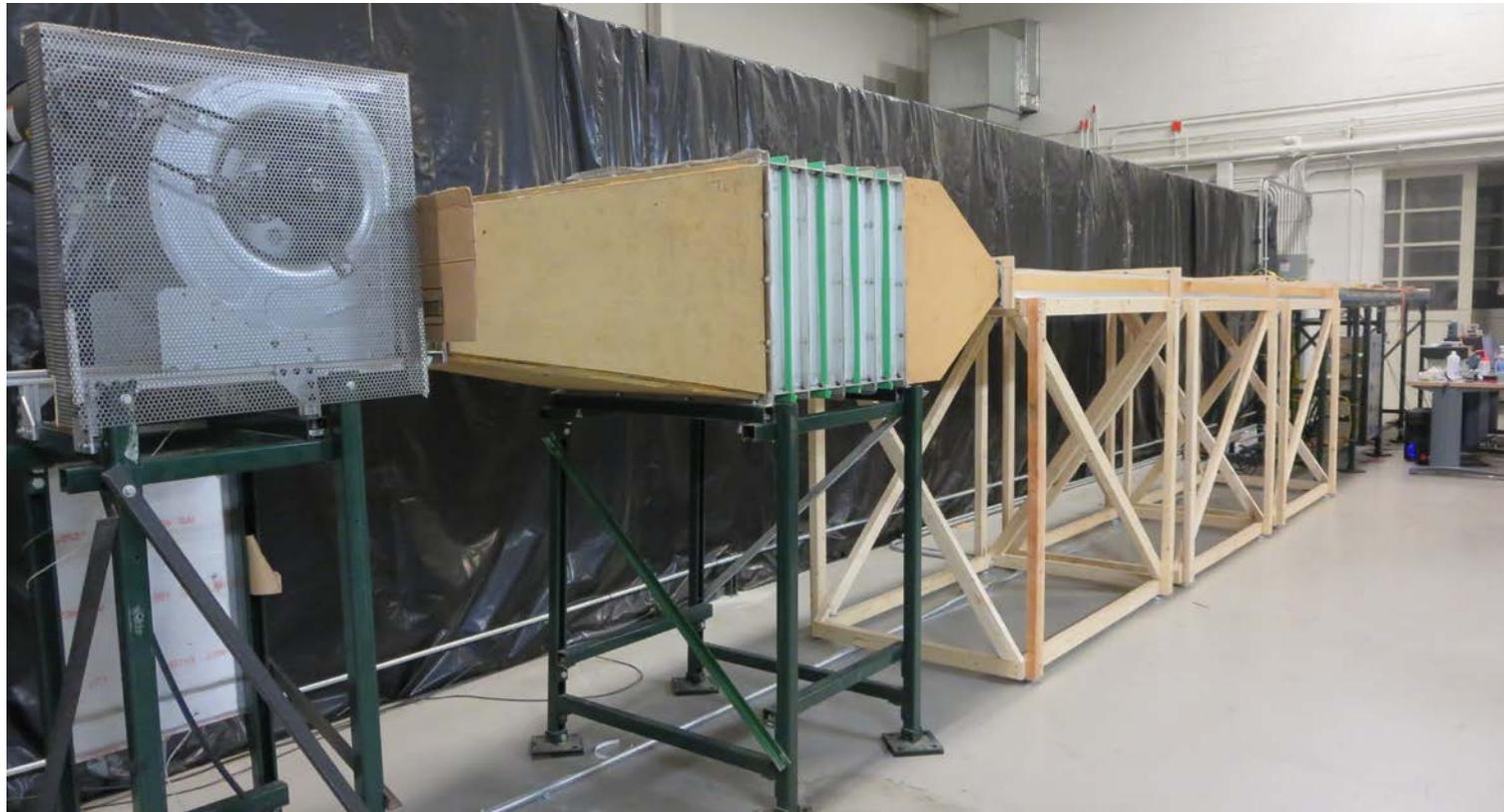
$$\frac{\partial \Theta}{\partial t} + U \frac{\partial \Theta}{\partial x} = \alpha \frac{\partial^2 \Theta}{\partial y^2} - \frac{\partial \overline{v' \theta'}}{\partial y} \leftarrow \text{Scalar flux}$$

- Scalar flux: contribution of turbulence in the transport of the scalar
- Mean quantities alone are **not enough** to understand scalar transport in turbulent flows
- Knowledge about the **scalar flux** is needed

# Two new facilities for temperature investigations

## Channel flow

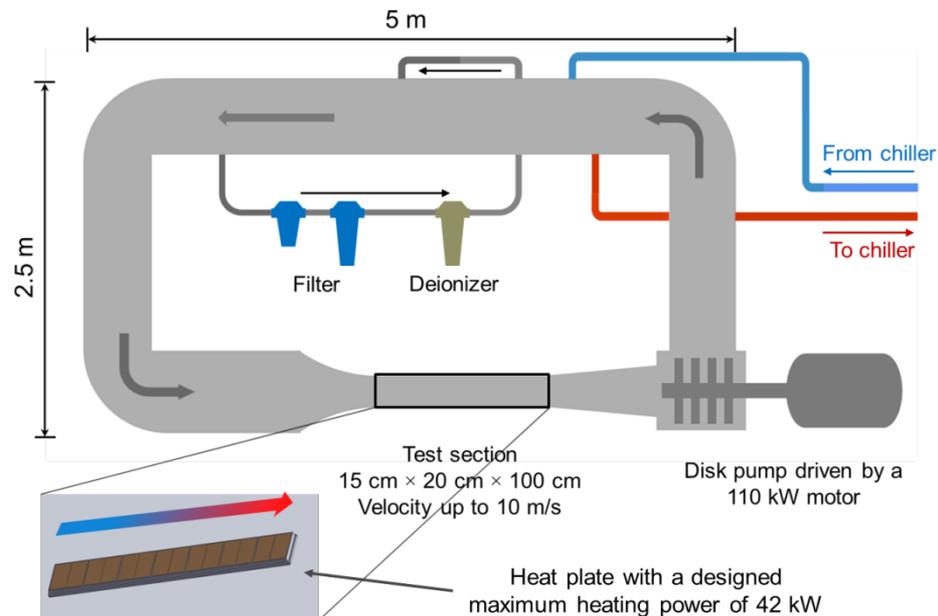
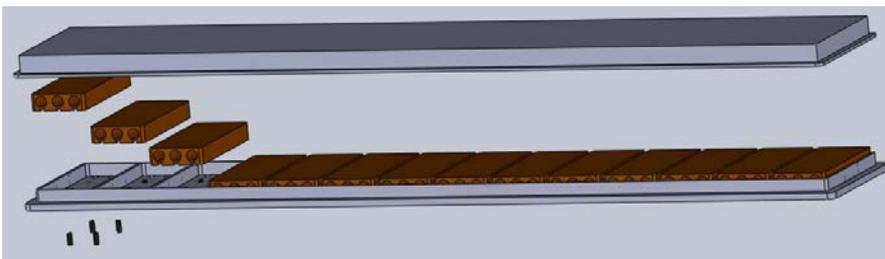
- Height  $h = 2\delta = 6.35\text{cm}$
- Aspect ratio 12
- Unheated section: 5 meters  $\sim 80h$
- Heated section: 4.75 meters  $\sim 75h$



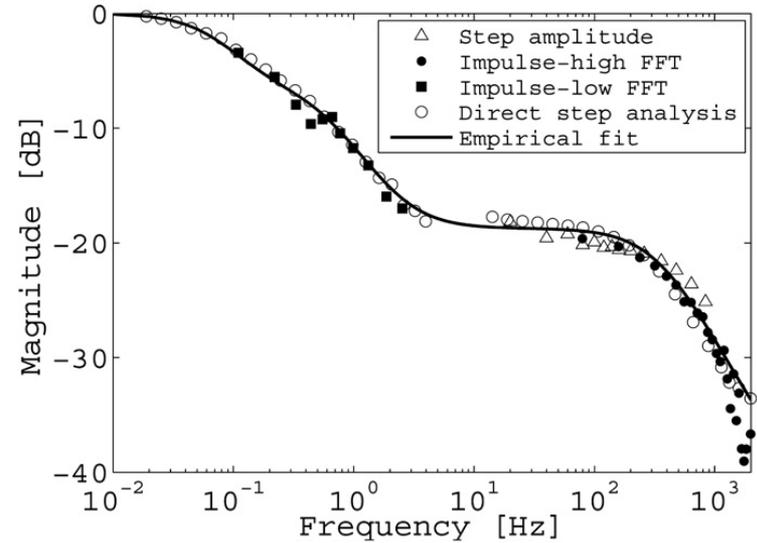
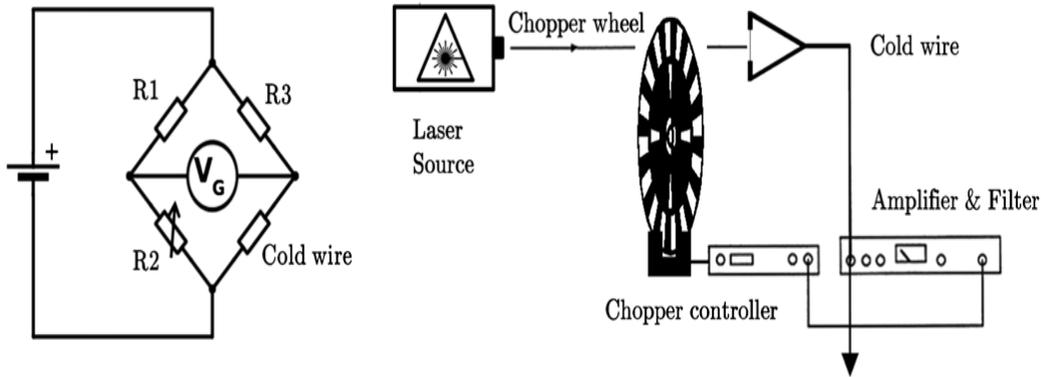
# Two new facilities for temperature investigations

## Water channel

- 0.25 x 0.15 x 1 m test section
- Up to 13 m/s
- 30 kW of cooling power installed
- 42 kW of heating built into the wall of the test section for a developing thermal and velocity boundary layer
- De-ionizer

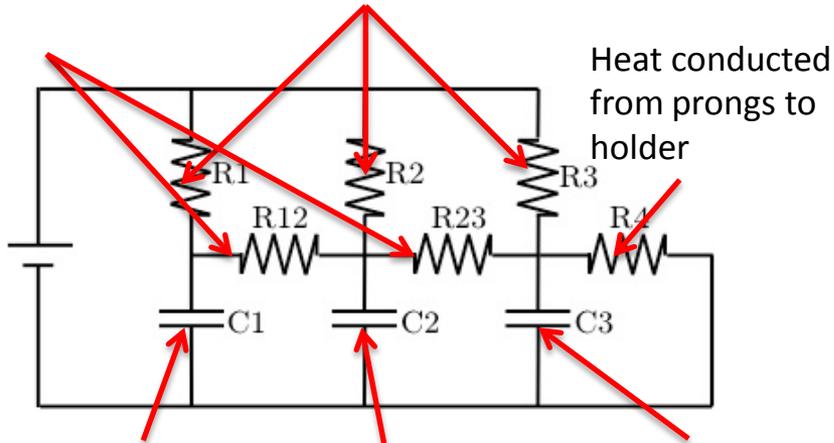


# Evaluating of cold-wires



Conduction between elements

Heat transfer

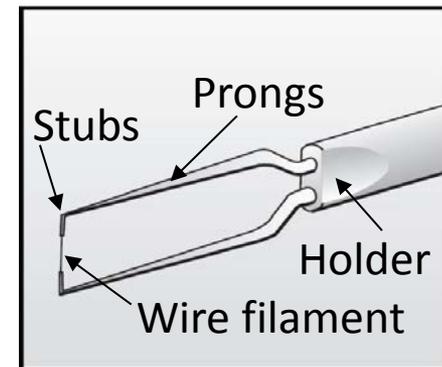


Heat accumulated in wire

Heat accumulated in stubs

Heat accumulated in prongs

Heat conducted from prongs to holder



# Design of true fast response temperature sensor

- Long and thin wire filament
- High conductivity prongs
- Thicker and shorter prongs
- Two metal construction  
platinum wire and gold stubs
- $200 \times 0.1 \times 1 \mu m$

Wire filament

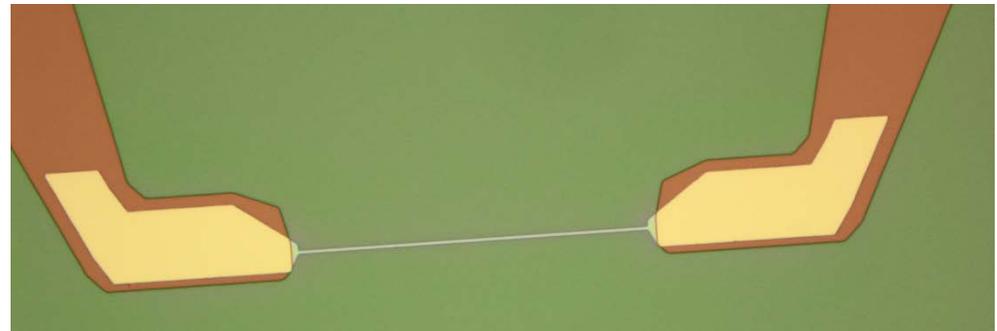
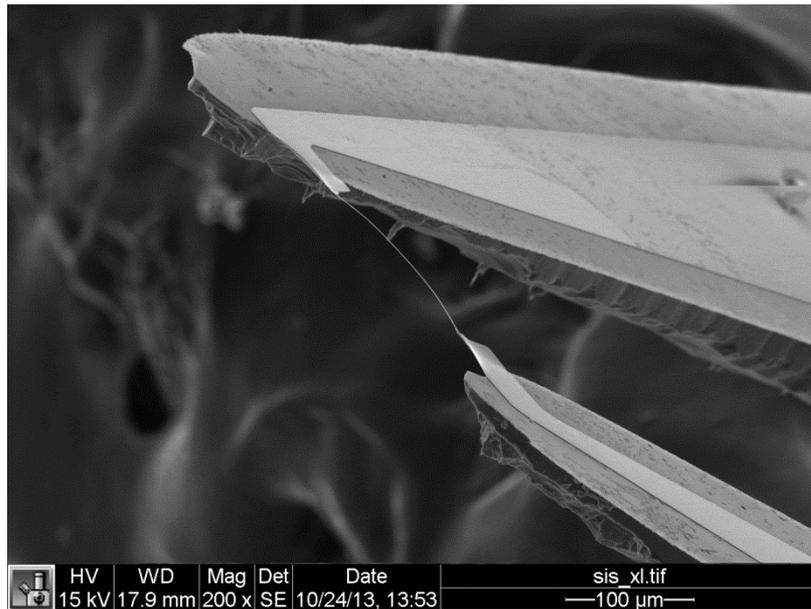
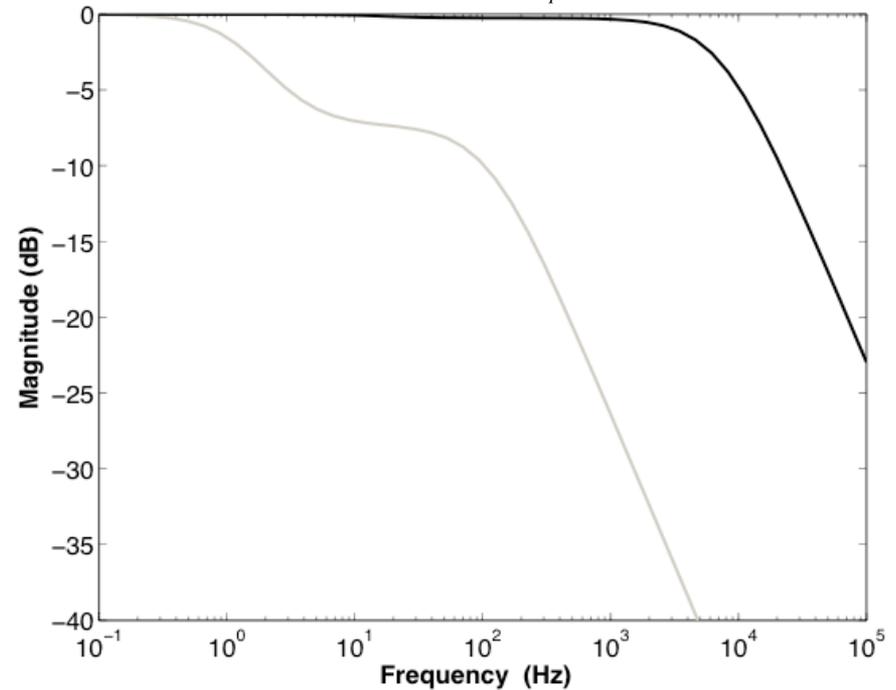
$$l = 300 \mu m$$

$$d = 200 \eta m$$

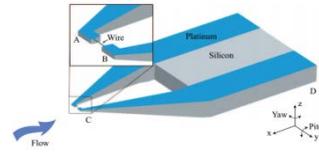
Gold prongs

$$l_p = 1 mm$$

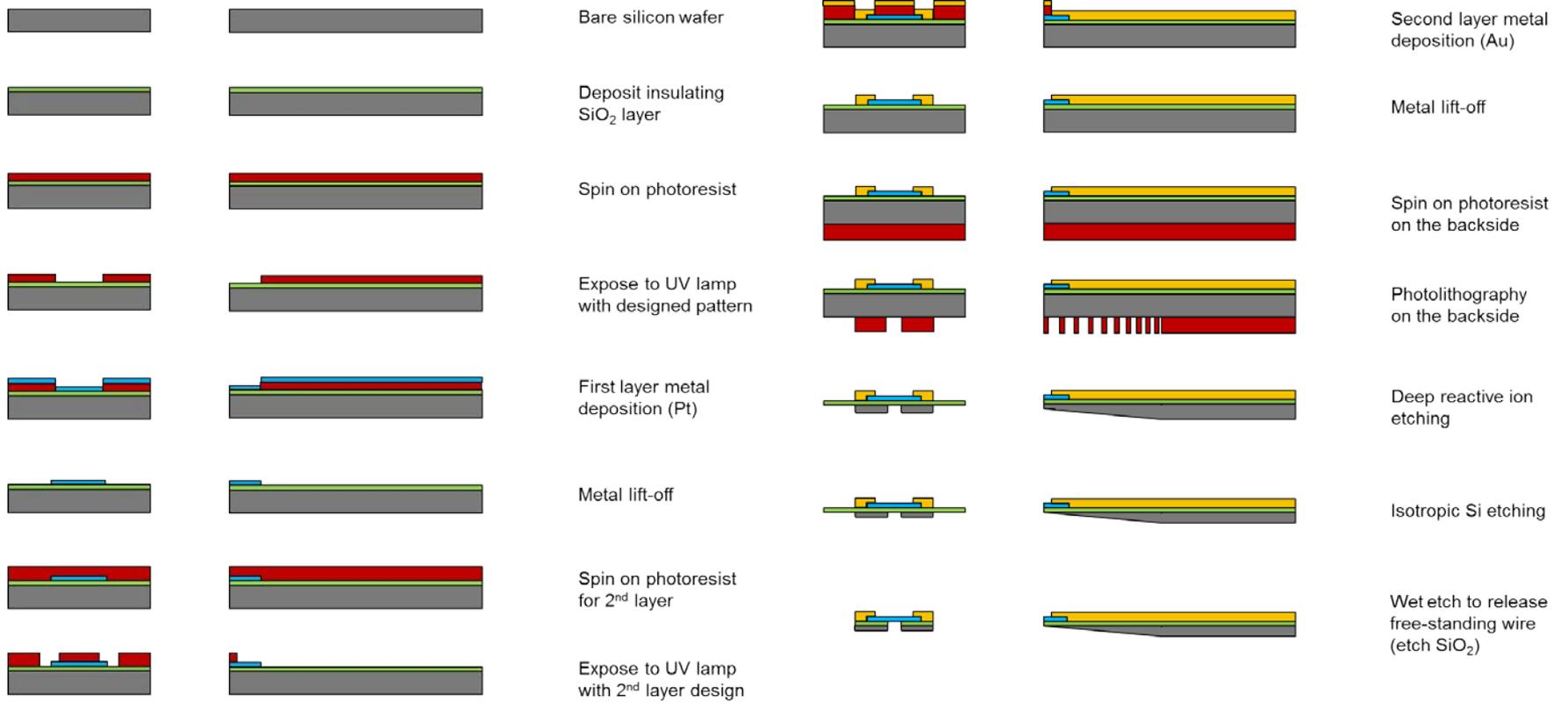
$$d_p = 1 mm$$



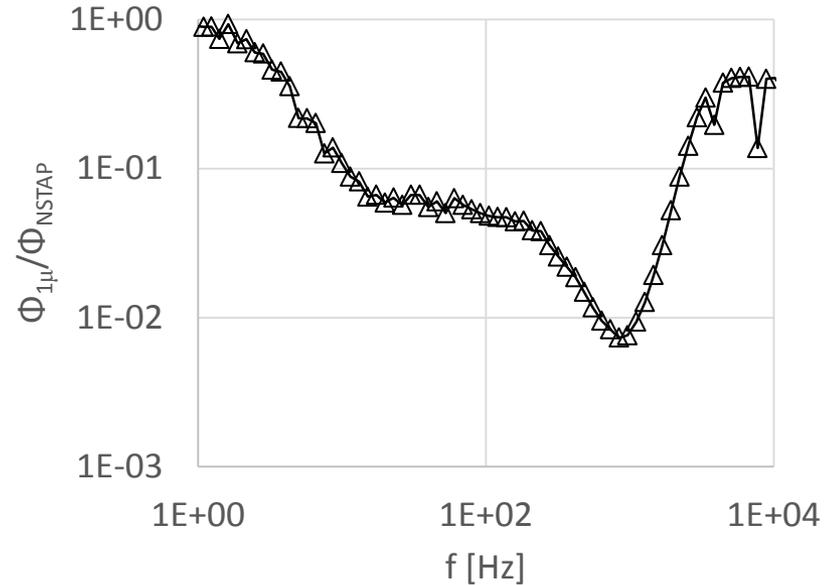
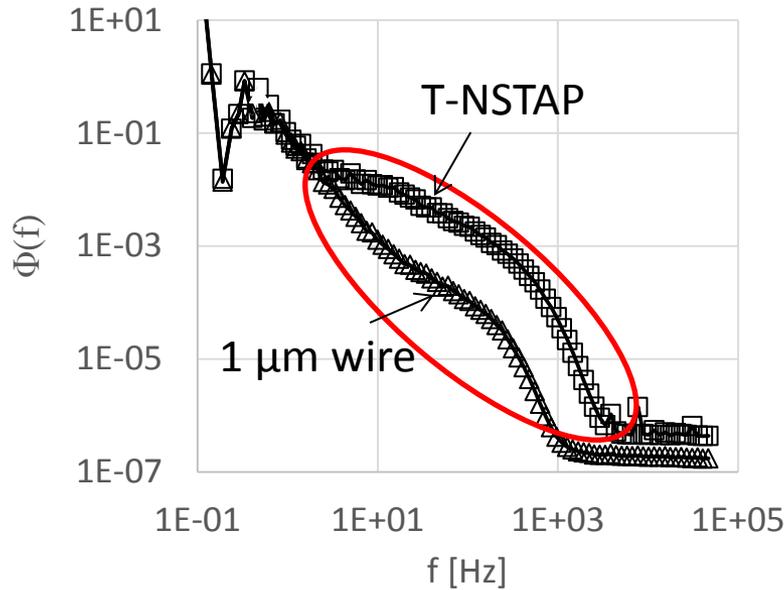
# Fabrication process



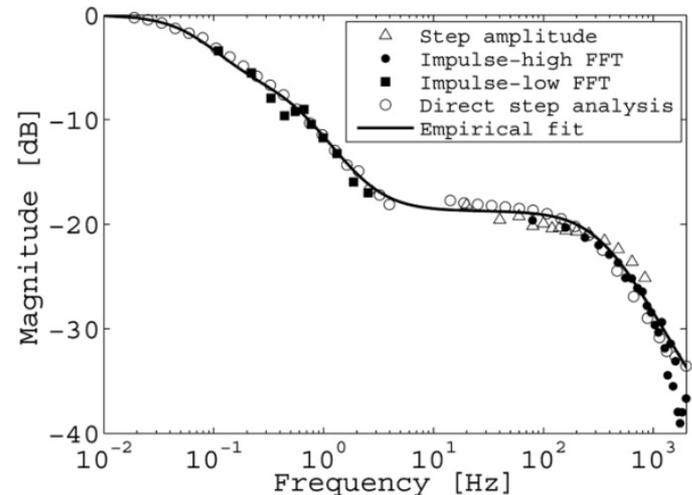
	Silicon		Photoresist
	Silicon dioxide		Platinum
			Gold



# Measurement techniques – avoiding attenuation and resolving dissipative scales



- Temperature fluctuations measured with
  - T-NSTAP (100 nm thickness)
  - 1  $\mu\text{m}$  wire ( $l/d=200$ )
- T-NSTAP has improved temporal and spatial resolution



# Conclusions

- A new theory for the streamwise turbulent fluctuations in fully developed pipe flow was introduced.
- The slope of the logarithm of the variances relate to the derivative in Reynolds number.
- To investigate similar behavior for scalar fields two new facilities have been commissioned.
- A new fast response cold-wire is developed and tested